

Static Structural Analysis of Boat Hull Using Different Materials

A. Prem Anandh

Department of Naval Architecture and Offshore Engineering, AMET University, Chennai, India

Abstract: A design and analysis has been developed to investigate a boat hull ship structure, hull structure is modelled by titanium nitrate frame panels. Wave loads and slamming loads acting on both hull structures have been calculated according to ABS rules at sea state with a ship velocity of 20 knots. Comparisons of deformations and stresses between two sets of loadings demonstrate that slamming loads have more detrimental effects on ship structure.

Key words: Boat hull, design, analysis, ABS and boat structure, loadings demonstrate, slamming loads

INTRODUCTION

Boat hull dynamic weights are weights that move in time with periods stretching out from a few seconds to a couple of minutes (Berger and Tinic, 1988). A vessel in development will moreover experience different dynamic stacks as the outcome of contraption vibrations and seaway joint efforts. These dynamic weights on ship structures can be segregated into two orders (Ostapenko and Vaucher, 1980). One is low repeat weights and other is high repeat loads. High repeat weights or typical vibration can be increased and transmitted by structure, structures exhibited to vibrations near their normal repeat can “get” the vibration (resonation) (Chen, 1982; Ostapenko and Chen, 1982). This may be felt on various vessels, for example as a shudder in a vessel as it stimulates and the engines rpm experiences the ordinary frequencies of assistant parts. High repeat dynamic weights increase evacuating and redirection of helper people, possibly provoking or enlivening depletion disillusionments (Ostapenko and Chen, 1982).

Disputed study, design of acoustic modem for autonomous underwater vehicles (Sathishkumar and Rajavel, 2014) exposes a design of bandwidth characteristics of the composite vehicle-manipulator dynamics. Design model on ship trajectory control using particle swarm optimization (Sethuramalingam and Nagaraj, 2015) audited by fuzzy model reference learning controller for ship trajectory (Paik *et al.*, 2001; Paik and Thayamballi, 2003). Design of optical sensor for detection of brininess of water (Lavanya *et al.*, 2014) predicting the salinity of water from the environment and checking it may be vital or harmful.

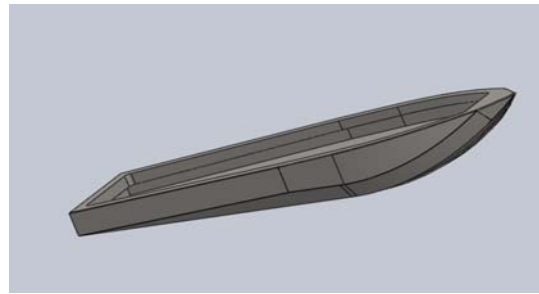


Fig. 1: Solid model of boat hull

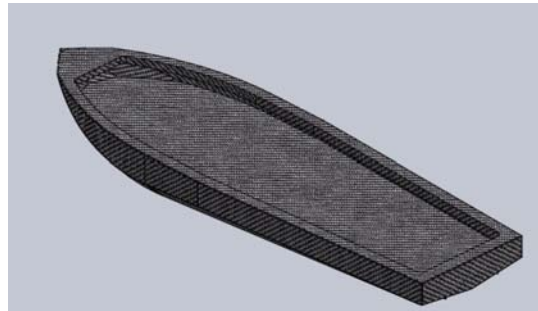


Fig. 2: Solid model model of boat hull top view

The boat hull is created as solid model by the SOLIDWORR Software. The dimensions are taken from presently used fishers boat. Figure 1 and 2 shows the solid model of the boat hull in two different views.

MATERIALS AND METHODS

The boat hull is analysis by the ANSYS research bench software using composite material for reduce the weight of boat. The boat hull solid model is created by the

SolidWork Software. In Table 1 and 2, all properties of the material are given. In this research, two materials are used for analyse the boat hull. One is natural fiber and another one is Si542.

Design and meshing of hull: The utilization of composites structure for submerged vehicle has been begun quite a while back at the Center for Unmanned System Studies (CentrUMS)-ITB2. The composite material has been utilized to assemble the primary structure (fundamental frame and floatation) and in addition for the fortified sub structure (apparatuses, controller) in CentrUMS-ITB’s submerged vehicles. The case of the composite-based submerged vehicle stage in our examination focus. The meshing model of the boat hull was shown in the Fig. 3.

On the off chance that contrasted and another metal structure, this structure tends to build the working profundity to weight proportion more than 2 times. The lesson gained from this experience is utilized for the new outline of half breed AUV Model.

Table 1: Material properties

Variables	Values
Volume	32.20 (mm ³)
Material	Fiber
Coordinates	Cartesian
Mass	68.25 (kg)
Nodes	641
Elements	635

Table 2: Mechanical properties

Variables	Values
Density	10.24 (g/cm ³)
Yield strength	52 (MPa)
Compressive strength	721 (MPa)
Modulus of elasticity	552 (GPa)
Vickers hardness	2584
Thermal conductivity	1.22

The present practices for the outline of submarine weight frames have been reviewed by MacKay in 3. Blend of exploratory and limited component demonstrating was distinguished as the essential practice in researching shell structure under outside weight. In the study, it was discovered that the exactness of an extensive variety of nonlinear numerical strategies including axis symmetric limited distinction and general shell limited.

Analysis of hull: In order to validate the specimen design prior to manufacturing and building the first prototype, a testing facility to make sure the satisfaction of the load requirement is needed. The facility to meet the entire requirement is pressure chamber.

The analysis result of the boat hull was shown in the Fig. 4. Figure 4 shows the von mises stress of the boat hull. This result was taken from the ANSYS research bench software. The maximum and minimum von mises stress points are denoted by the colours.

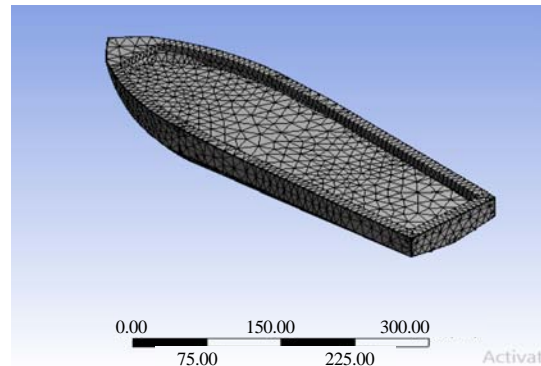


Fig. 3: Meshing model model of boat hull

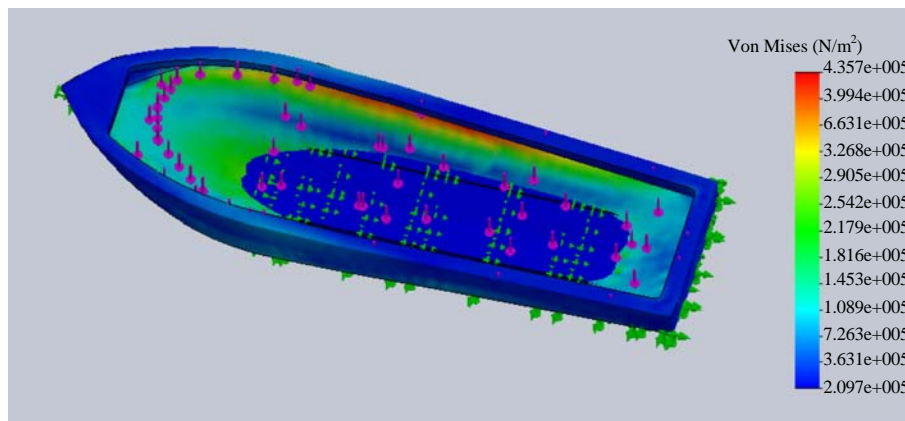


Fig. 4: Von Mises stress

Table 3: Results

Materials	Results		
	Stress	Displacement	Sliding contact
Fibre	13.12	0.43e-3	0.201e-5
Si542	6.23	0.21e-3	0.205e-4

RESULTS AND DISCUSSION

With the post processing function of the finite element analysis, some result and performance prediction of this composites hull. The results of the both fiber and Si542 are noted in the Table 3. The Si542 has less stress value than the natural fiber. The Si542 has 6.32 N/mm² stress only but the natural fiber has 13.12 N/mm² stress. Compared with the natural fiber the stress value was double of the Si542.

CONCLUSION

All strategies that required on this arrangement of investigation for composites body indicated promising outcomes. With some emphasis on number and heading of layer. The material properties for weight chamber. With a specific end goal to approve the example outline before assembling and building the main model, a testing office to ensure the fulfilment of the heap prerequisite is required.

REFERENCES

Berger, E. and S. Tinic, 1988. Lessons learned from full-scale vibration tests on nuclear power plant auxiliary structure in Switzerland. International Atomic Energy Agency, 35: 129-140.

Chen, Y., 1982. Strength of ship hull girders under moment, shear and torque. Msc Thesis, Lehigh University, Bethlehem, Pennsylvania.

Lavanya, J., S.K. Roy and P. Sharan, 2014. Design of optical sensor for detection of brininess of water. Proceedings of the Conference on Global Humanitarian Technology South Asia Satellite (GHTC-SAS), September 26-27, 2014, IEEE, Trivandrum, India, ISBN:978-1-4799-4097-4, pp: 99-104.

Ostapenko, A. and A. Vaucher, 1980. Ultimate strength of ship hull girders under moment, shear and torque. Master Thesis, Lehigh University, Bethlehem, Pennsylvania.

Ostapenko, A.L.E.X.I.S. and Y. Chen, 1982. Effect of Torsion on Strength of Ship Hulls. Lehigh University, Bethlehem, Pennsylvania, Pages: 76.

Paik, J.K. and A.K. Thayamballi, 2003. A concise introduction to the idealized structural unit method for nonlinear analysis of large plated structures and its application. Thin-Walled Struct., 41: 329-355.

Paik, J.K., A.K. Thayamballi, P.T. Pedersen and Y.I. Park, 2001. Ultimate strength of ship hulls under torsion. Ocean Eng., 28: 1097-1133.

Sathishkumar, R. and R. Rajavel, 2014. Design of acoustic modem for an autonomous underwater vehicles. Intl. J. Appl. Eng. Res., 9: 10123-10135.

Sethuramalingam, T.K. and B. Nagaraj, 2015. Design model on ship trajectory control using particle swarm optimisation. Proceedings of the 2015 Online International Conference on Green Engineering and Technologies, November 27-27, 2015, IEEE, Coimbatore, India, ISBN:978-1-4673-9781-0, pp: 1-6.